

CASES OF ASME BOILER AND PRESSURE VESSEL CODE

Approval Date: August 14, 1997

*See Numeric Index for expiration
and any reaffirmation dates.*

**Case N-513
Evaluation Criteria for Temporary Acceptance of
Flaws in Class 3 Piping
Section XI, Division 1**

Inquiry: What rules may be used for temporary acceptance of flaws, including through-wall flaws, in moderate energy Class 3 piping, without repair or replacement?

Reply: It is the opinion of the Committee that the following rules may be used to accept flaws, including through-wall flaws, in moderate energy Class 3 piping, without repair or replacement for a limited time, not exceeding the time to the next scheduled outage.

1.0 SCOPE

(a) These requirements apply to the ASME Section III, ANSI B31.1, and ANSI B31.7 piping, classified by the Owner as Class 3.

(b) The provisions of the Case apply to Class 3 piping whose maximum operating temperature does not exceed 200°F and whose maximum operating pressure does not exceed 275 psig.

(c) The following flaw evaluation criteria are permitted for pipe and tube. The flaw evaluation criteria are permitted for adjoining fittings and flanges to a distance of $(R_o t)^{1/2}$ from the weld centerline.

(d) The provisions of this Case demonstrate the integrity of the item and not the consequences of leakage. It is the responsibility of the Owner to demonstrate system operability considering effects of leakage.

2.0 PROCEDURE

(a) The flaw geometry shall be characterized by volumetric inspection methods or by physical measurement. The full pipe circumference at the flaw location shall be inspected to characterize the length and depth of all flaws in the pipe section.

(b) Flaw shall be classified as planar or nonplanar.

(c) When multiple flaws, including irregular (compound) shape flaws, are detected, the interaction and combined area loss of flaws in a given pipe section shall be accounted for in the flaw evaluation.

(d) A flaw evaluation shall be performed to determine the conditions for flaw acceptance. Section 3.0 provides accepted methods for conducting the required analysis.

(e) Frequent periodic inspections of no more than 30 day intervals shall be used to determine if flaws are growing and to establish the time at which the detected flaw will reach the allowable size. Alternatively, a flaw growth evaluation may be performed to predict the time at which the detected flaw will grow to the allowable size. When a flaw growth analysis is used to establish the allowable time for temporary operation, periodic examinations of no more than 90 day intervals shall be conducted to verify the flaw growth analysis predictions.

(f) For through-wall leaking flaws, leakage shall be observed by daily walkdowns to confirm the analysis conditions used in the evaluation remain valid.

(g) If examinations reveal flaw growth rate to be unacceptable, a repair or replacement shall be performed.

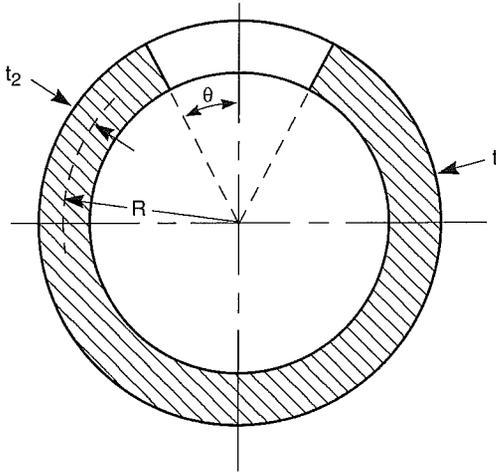
(h) Repair or replacement shall be performed no later than when the predicted flaw size from either periodic inspection or by flaw growth analysis exceeds the acceptance criteria of Section 4.0, or the next scheduled outage, whichever occurs first. Repair or replacement shall be in accordance with IWA-4000 or IWA-7000, respectively, in Editions and Addenda prior to the 1991 Addenda; and, in the 1991 Addenda and later, in accordance with IWA-4000.

(i) Evaluations and examination shall be documented in accordance with IWA-6300. The Owner shall document the use of this Case on the applicable data report form.

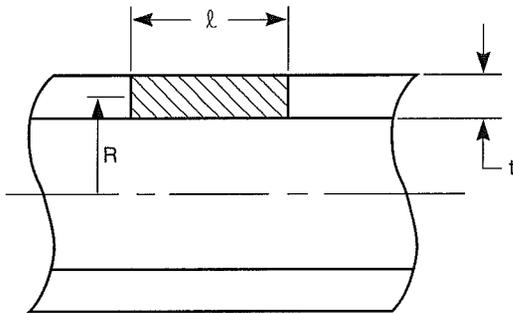
3.0 FLAW EVALUATION

(a) For planar flaws, the flaw shall be bounded by a rectangular or circumferential planar area in accordance with the methods described in Appendix C or Appendix H. IWA-3300 shall be used to determine when multiple proximate flaws are to be evaluated as a single flaw. The geometry of a through-wall planar flaw is shown in Fig. 1.

(b) For planar flaws in austenitic piping, the evaluation procedure in Appendix C of Section XI, Division 1, shall be used. Flaw depths up to 100% of wall



(a) Circumferential Flaw



(b) Axial Flaw

FIG. 1 THROUGH-WALL FLAW GEOMETRY

thickness may be evaluated. When through-wall circumferential flaws are evaluated, the formulas for evaluation given in Articles C-3320 of Appendix C may be used, with the flaw penetration (a/t) equal to unity. When through-wall axial flaws are evaluated, the allowable flaw length is:

$$\ell_{all} = 1.58\sqrt{Rt} \left[\left(\frac{\sigma_f}{(SF)\sigma_h} \right)^2 - 1 \right]^{1/2} \quad (1)$$

$$\sigma_h = pD_o/2t \quad (2)$$

$$\sigma_f = (S_y + S_u)/2 \quad (3)$$

where

p = is pressure for the loading condition

D_o = is pipe outside diameter

σ_f = is the flow stress

S_y = is the code yield strength

S_u = is the code tensile strength and

SF = is the safety factor as specified in C-3420 of Appendix C

Material properties at the temperature of interest shall be used.

(c) For planar flaws in ferritic piping, the evaluation procedure in Article H-7000 of Appendix H, Section XI, Division 1, shall be used. Flaw depths up to 100% of wall thickness may be evaluated. When through-wall flaws are evaluated, the formulas for evaluation given in Articles H-7300 and H-7400 of Appendix H may be used, but with values for F_m , F_b , and F applicable to through-wall flaws. Relations for F_m , F_b , and F that take into account flaw shape and pipe geometry (R/t ratio) shall be used. The appendix to this Code Case provides equations for F_m , F_b , and F for a selected range of R/t . Geometry of a through-wall crack is shown in Fig. 1.

(d) For nonplanar flaws, the pipe is acceptable when the remaining pipe thickness (t_p) is greater than or equal to the minimum wall thickness t_{min} :

$$t_{min} = \frac{pD_o}{2(S + 0.4p)} \quad (4)$$

where

p = is the maximum operating pressure at flaw location

S = is the allowable stress at operating temperature

Where appropriate, bending load at the flaw location shall be considered in the determination of t_{min} . When t_p is less than t_{min} , an evaluation shall be performed as given below. The evaluation procedure is a function of the depth and the extent of the affected area as illustrated in Fig. 2.

(1) When the width of wall thinning that exceeds t_{min} , W_m , is less than or equal to $0.5 (R_o t_{min})^{1/2}$, where R_o is the outside radius and W_m is defined in Fig. 2, the flaw can be classified as a planar flaw and evaluated under para. 3(a) through para. 3(c). When the above requirement is not satisfied, (2) shall be met.

(2) When the transverse extent of wall thinning that exceeds t_{min} , $L_{m(t)}$, is not greater than $(R_o t_{min})^{1/2}$,

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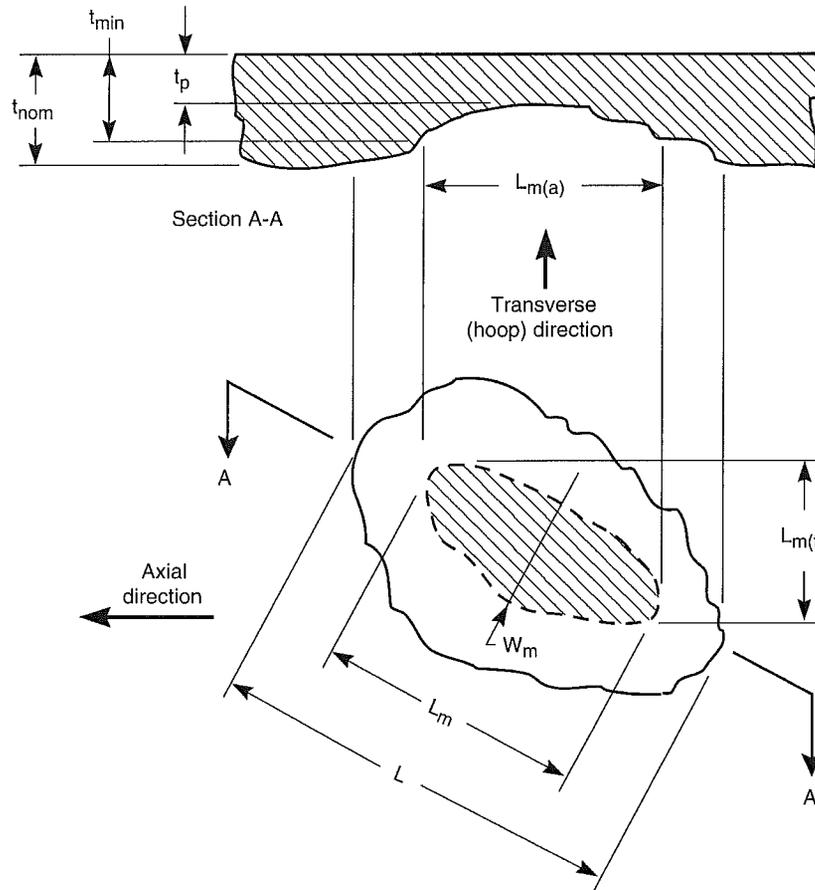


FIG. 2 ILLUSTRATION OF NONPLANAR FLAW DUE TO WALL THINNING

t_{aloc} is determined from Curve 1 of Fig. 3, where $L_{m(t)}$ is defined in Fig. 2. When the above requirement is not satisfied, (3) shall be met.

(3) When the maximum extent of wall thinning that exceeds t_{min} , L_m , is less than or equal to $2.65 (R_o t_{min})^{1/2}$ and t_{nom} is greater than $1.13 t_{min}$, t_{aloc} is determined by satisfying both of the following equations:

$$\frac{t_{aloc}}{t_{min}} \geq \frac{1.5 \sqrt{R_o t_{min}}}{L} \left[1 - \frac{t_{nom}}{t_{min}} \right] + 1.0 \quad (5)$$

$$\frac{t_{aloc}}{t_{min}} \geq \frac{0.353 L_m}{\sqrt{R_o t_{min}}} \quad (6)$$

When the above requirements are not satisfied, (4) shall be met.

(4) When the requirements of (1), (2) and (3) above are not satisfied, t_{aloc} is determined from Curve 2 of Fig. 3. In addition, t_{aloc} shall satisfy the following equation:

$$\frac{t_{aloc}}{t_{min}} \geq \frac{0.5 + \left(\frac{t_{nom}}{t_{min}} \right) \left(\frac{\sigma_b}{S} \right)}{1.8} \quad (7)$$

where σ_b is the nominal pipe longitudinal bending stress resulting from all primary pipe loadings.

(e) For nonferrous materials, nonplanar and planar flaws may be evaluated following the general approach

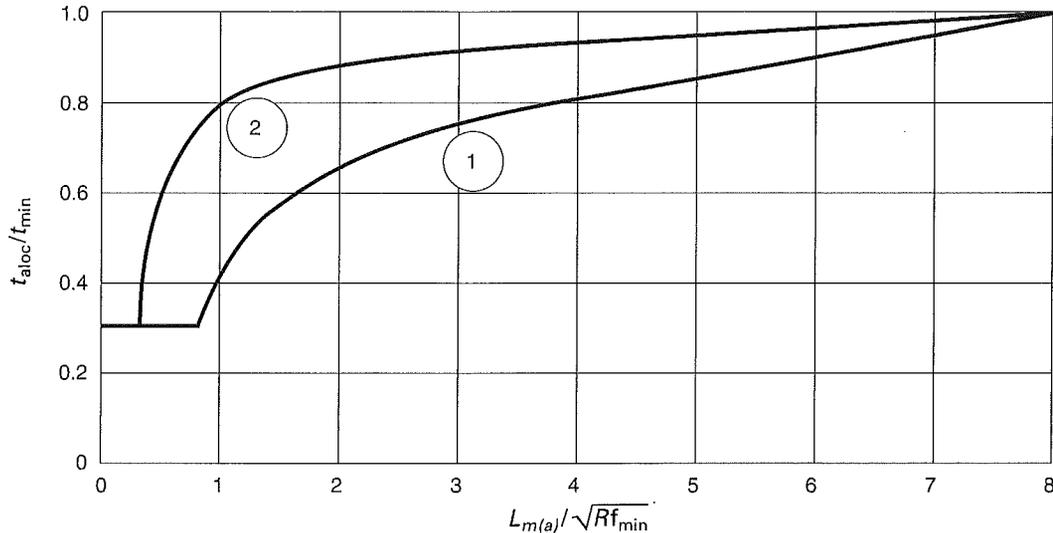


FIG. 3 ALLOWABLE WALL THICKNESS AND LENGTH OF LOCALLY THINNED AREA

of (a) through (d) above. For ductile materials, the approach given in (b) may be used; otherwise, the approach given in (c) and (d) should be applied. Safety factors provided in Section 4.0 shall be used. It is the responsibility of the evaluator to establish conservative estimates of strength and fracture toughness for the piping material.

4.0 ACCEPTANCE CRITERIA

The piping containing a circumferential planar flaw is acceptable for continued temporary service when flaw evaluation provides a safety margin, based on load, of a factor of 2.77 for Level A and B and 1.39 for Level C and D service loading conditions. Piping containing a nonplanar flaw is acceptable for continued temporary service if $t_p \geq t_{aloc}$, where t_{aloc} is determined from Section 3(d).

Lower safety factors may be used, provided that a detailed engineering evaluation of continued operation demonstrates that lower safety factors are justified.

5.0 AUGMENTED EXAMINATION

An augmented volumetric examination or physical measurement to assess degradation of the affected system shall be performed as follows:

(a) From the engineering evaluation, the most susceptible locations shall be identified. A sample size of at least five of the most susceptible and accessible locations, or, if fewer than five, all susceptible and accessible locations shall be examined within 30 days of detecting the flaw.

(b) When a flaw is detected, an additional sample of the same size as defined in 5(a) shall be examined.

(c) This process shall be repeated within 15 days for each successive sample, until no significant flaw is detected or until 100% of susceptible and accessible locations have been examined.

6.0 NOMENCLATURE

- c = half crack length
- D_o = outside pipe diameter
- F = nondimensional stress intensity factor for through-wall axial flaw under hoop stress
- F_b = nondimensional stress intensity factor for through-wall circumferential flaw under pipe bending stress
- F_m = nondimensional stress intensity factor for through-wall circumferential flaw under membrane stress
- ℓ = total crack length = $2c$
- ℓ_{all} = allowable axial through-wall length

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- L = maximum extent of a local thinned area with $t < t_{nom}$
 L_m = maximum extent of a local thinned area with $t < t_{min}$
 $L_{m(a)}$ = axial extent of wall thinning below t_{min}
 $L_{m(t)}$ = circumferential extent of wall thinning below t_{min}
 p = maximum operating pressure at flaw location
 R = mean pipe radius
 R_o = outside pipe radius
 S = allowable stress at operating temperature
 S_u = code specified ultimate tensile strength
 S_y = code specified yield strength
 t = wall thickness
 t_{alloc} = allowable local thickness for a nonplanar flaw that exceeds t_{min}
 t_{min} = minimum wall thickness required for pressure loading
 t_{nom} = nominal wall thickness
 t_p = minimum remaining wall thickness
 W_m = maximum extent of a local thinned area perpendicular to L_m with $t < t_{min}$
 λ = nondimensional half crack length for through-wall axial flaw
 σ_f = material flow stress
 σ_h = pipe hoop stress due to pressure
 σ_b = nominal longitudinal bending stress for primary loading without stress intensification factor
 Θ = half crack angle for through-wall circumferential flaw

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APPENDIX I

RELATIONS FOR F_m , F_b , AND F FOR THROUGH-WALL FLAWS

I-1.0 DEFINITIONS

For through-wall flaws, the crack depth (a) will be replaced with half crack length (c) in the stress intensity factor equations in Articles H-7300 and H-7400 of Section XI, Appendix H. Also, Q will be set equal to unity in Article H-7400.

I-2.0 CIRCUMFERENTIAL FLAWS

For a range of R/t between 5 and 20, the following equations for F_m and F_b may be used:

$$F_m = 1 + A_m (\Theta/\pi)^{1.5} + B_m (\Theta/\pi)^{2.5} + C_m (\Theta/\pi)^{3.5}$$

$$F_b = 1 + A_b (\Theta/\pi)^{1.5} + B_b (\Theta/\pi)^{2.5} + C_b (\Theta/\pi)^{3.5}$$

where

Θ = Half crack angle = c/R

R = Mean pipe radius

t = Pipe wall thickness

and

$$A_m = -2.02917 + 1.67763 (R/t) - 0.07987 (R/t)^2 + 0.00176 (R/t)^3$$

$$B_m = 7.09987 - 4.42394 (R/t) + 0.21036 (R/t)^2 - 0.00463 (R/t)^3$$

$$C_m = 7.79661 + 5.16676 (R/t) - 0.24577 (R/t)^2 + 0.00541 (R/t)^3$$

$$A_b = -3.26543 + 1.52784 (R/t) - 0.072698 (R/t)^2 + 0.0016011 (R/t)^3$$

$$B_b = 11.36322 - 3.91412 (R/t) + 0.18619 (R/t)^2 - 0.004099 (R/t)^3$$

$$C_b = -3.18609 + 3.84763 (R/t) - 0.18304 (R/t)^2 + 0.00403 (R/t)^3$$

Equations for F_m and F_b are accurate for R/t between 5 and 20 and become increasingly conservative for R/t greater than 20. Alternative solutions for F_m and F_b may be used when R/t is greater than 20.

I-3.0 AXIAL FLAWS

For internal pressure loading, the following equation for F may be used:

$$F = 1 + 0.072449\lambda + 0.64856\lambda^2 - 0.2327\lambda^3 + 0.038154\lambda^4 - 0.0023487\lambda^5$$

where

$$\lambda = c/(Rt)^{1/2}$$

c = half crack length

The equation for F is accurate for λ between 0 and 5. Alternative solutions for F may be used when λ is greater than 5.